

AMENDMENT TO THE SPECIFICATION

Page 3, replace the paragraph starting at line 14 through line 3 as follows:

In a subassembly in accordance with the invention, ~~with the characteristics given in the characterizing part of Claim 1,~~ because the counter-ring is no longer a separate part that must be inserted into the cover plate, less effort is required for installation. Because the counter-ring itself need no longer be so stable that it can be manipulated and mounted as a separate component, it can be made considerably less thick than conventional counter-rings, which for example must be pressed into a recess. The diminished thickness of the counter-ring offers two advantages. First, it reduces the length of the whole structure. Second, it improves the heat dissipation. That is, although such counter-rings are customarily made of material with extremely poor thermal conductivity, because the counter-ring is less thick it can adequately conduct heat away from the component into which it is integrated. The thermal conductivity is also improved in comparison with state-of-the-art constructions in that the counter-ring is now directly connected to the relevant component rather than being partially insulated therefrom, for instance by O-rings that must be pressed into place to serve as seals for the counter-ring according to the state of the art.

Page 6, replace the paragraph starting at line 1 through line 20 as follows:

An axial ~~piston~~ piston compressor in accordance with the invention ~~as described in Claim 14~~ offers the advantage that it is possible for the frictional heat that is generated to be especially well transferred from the counter-ring directly into the housing of the compressor. This reduces the warming of the axial face seal, which results in less thermal and mechanical distortion and deformation. Furthermore, installation is simplified because the counter-ring does not need to be separately inserted into the cover plate. Here the term 'cover plate' is understood to mean the part of the housing that is provided with the exit opening for the drive shaft; depending on the structural design and nature of mounting of the compressor, this part can also be the central part of the housing. Economical advantages are also obtained, because in comparison to the state of the art

two parts are eliminated, namely the counter-ring as a separate component and the O-ring that would otherwise be needed as a seal for the counter-ring. If leakage should occur in the region of the axial face seal, a new counter-ring can be installed with little effort simply by exchanging the cover plate.

Page 7, replace the paragraph starting at line 1 through line 16 as follows:

In Figure 1 an axial piston compressor according to the state of the art is shown. It contains a housing 10 in which a drive shaft 12 is rotatably mounted. Seated on the drive shaft is a wobble plate 14 that cooperates with pistons 16. In Figure 1 only a single piston is visible; in fact, ~~a~~as many as seven pistons are provided, each of which is translationally displaceable within a cylinder. In the region where the drive shaft 12 emerges from the housing 10 there is disposed a cover plate 18 provided with an exit aperture 19 through which the drive shaft 12 passes out of the housing 10. Pressed into the cover plate is a counter-ring 20 with an O-ring 21. A sliding ring 22 makes contact with the counter-ring 20 and is pressed against the counter-ring by a spring 24. By this means an axial face seal is formed, which seals off the housing from the exterior space in the region of the exit aperture 19.